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Disbond/Delamination Arrest Features in Aircraft Composite Structures

2013 Technical Review

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Project Information

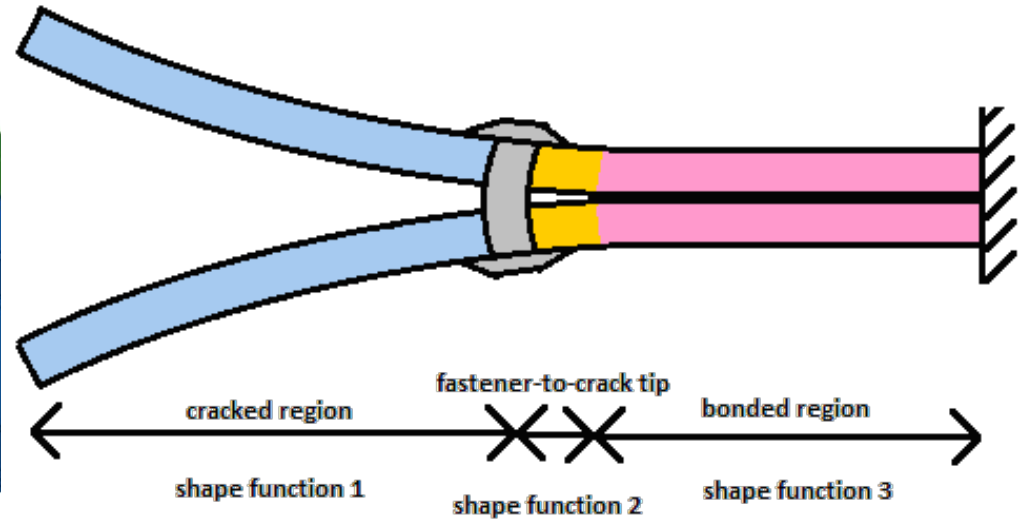
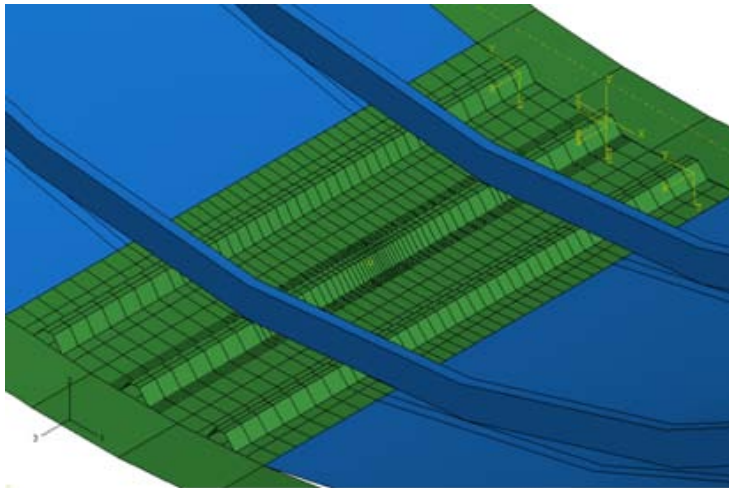
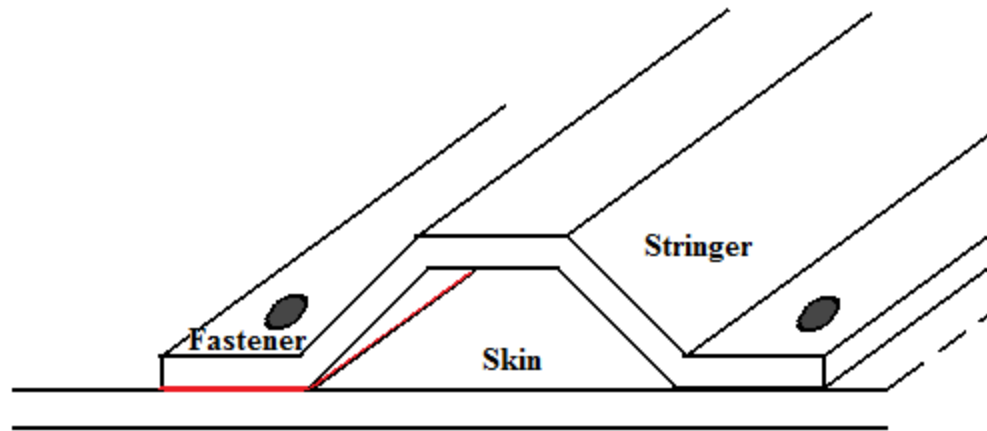
- Principal Investigator and Researchers (UW)
 - Prof. Kuen Y. Lin (PI)
 - Eric Cheung (Ph.D. student)
 - Wendy Liu (MS student)
 - Luke Richard (MS student)
- FAA Technical Monitor
 - Lynn Pham, Curtis Davies
- Other FAA Personnel Involved
 - Larry Ilcewicz
- Industry Participation
 - **Boeing:** Marc Piehl, Gerald Mabson, Eric Cregger, Matthew Dilligan (All from BR&T)
 - **Toray:** Kenichi Yoshioka, Don Lee, Felix Nguyen



Background

- Motivation and Key Issues
 - Delamination is one of the key issues for laminated and “bonded” composite structures
- Objectives
 - To understand the effectiveness of delamination/disbond arrest features
 - To develop analysis tools for design and optimization
- Approach
 - Construct FEM models in ABAQUS with VCCT
 - Perform sensitivity studies on fastener effectiveness
 - Conduct coupon-level experiments using novel specimens
 - Develop analytical tools validated by FEM and test

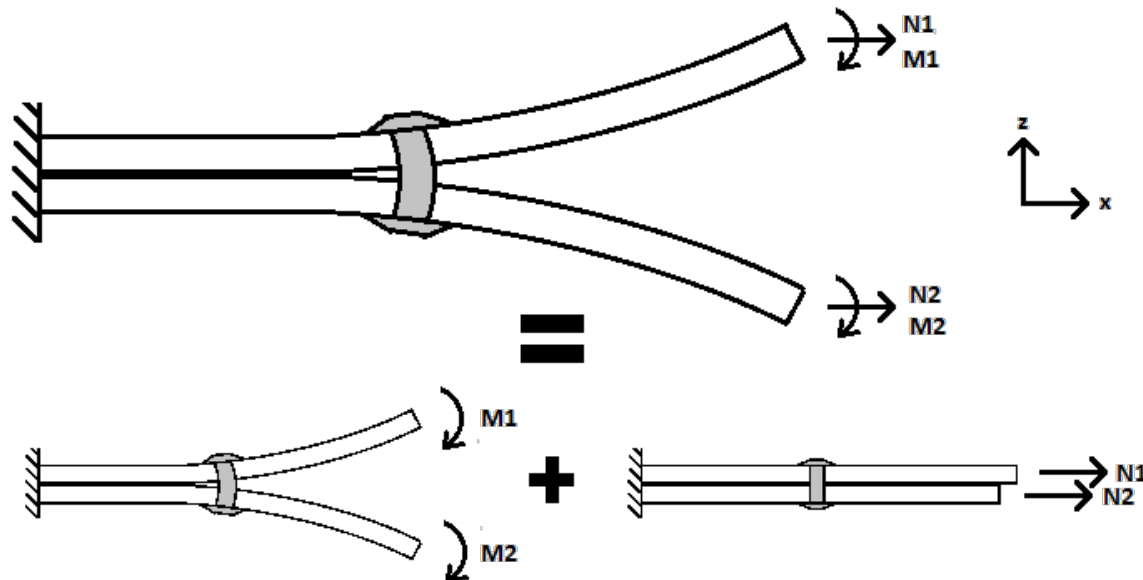
Crack Arrest Mechanism by Fastener





Analytical Model Development

- Model is composed of a beam-column part and a truss part
- Fastener is modeled by a tension spring which works with the beam-columns in bending; and a joint flexibility spring which works with the trusses
- Crack tip ERR is obtained using VCCT
- Friction and joint/hole clearance is also modeled





Beam-Column

- Polynomial shape function

$$w_i(x) = \sum_{j=0}^n \beta_{i,j} x^j$$

- Beam-Column energy

$$U_{bc,i} = \frac{1}{2} EI \int_{L_1}^{L_2} \left(\frac{d^2 w_i}{dx^2} \right)^2 dx + \frac{1}{2} N \int_{L_1}^{L_2} \left(\frac{dw_i}{dx} \right)^2 dx$$

Truss

- Polynomial shape function

$$u_i(x) = \sum_{j=0}^n \alpha_{i,j} x^j + \sum_{k=n+1}^m \alpha_{i,k} e^{c_k(x-L)}$$

- Truss energy

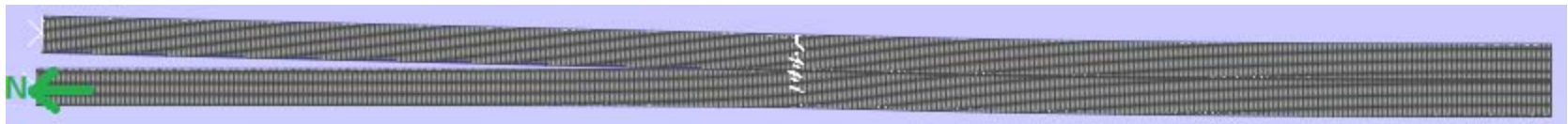
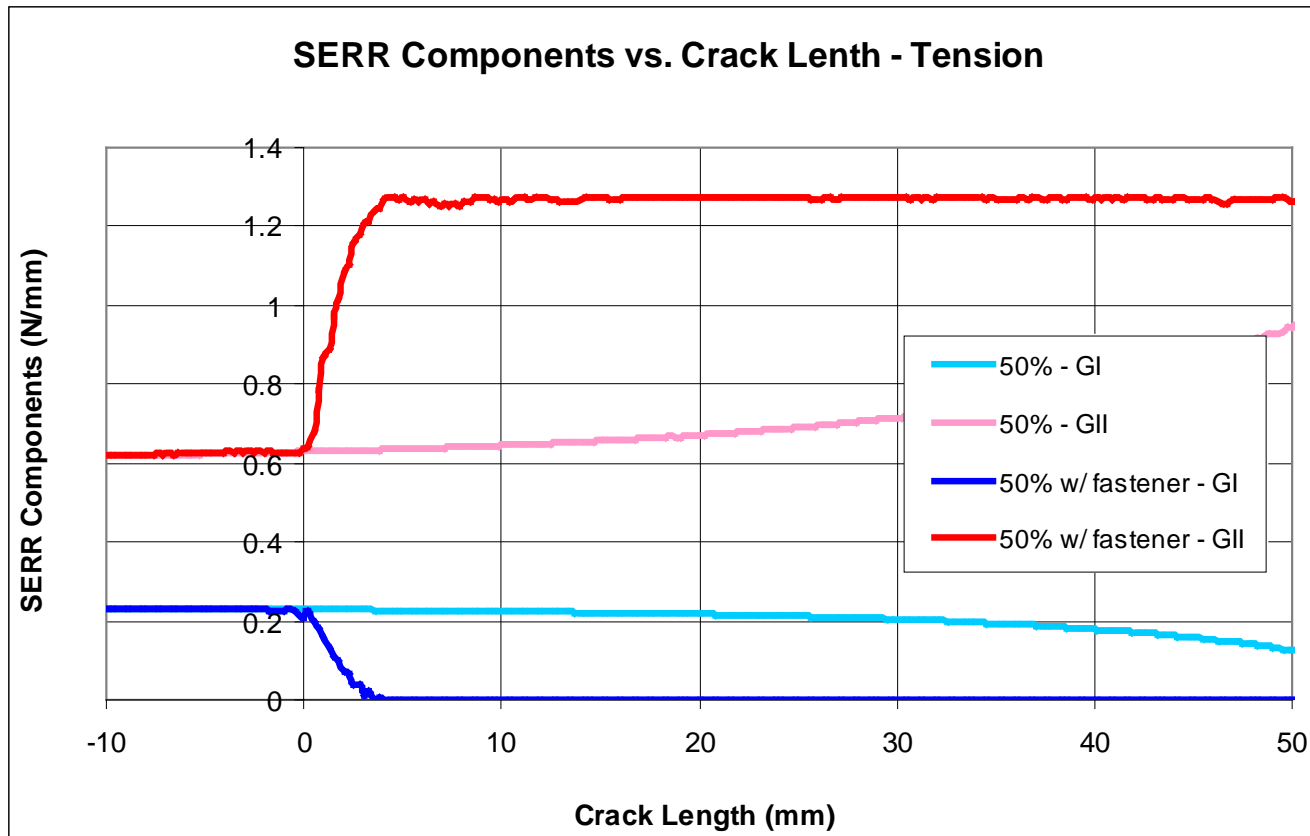
$$U_{truss,i} = \frac{1}{2} AE \int_{L_1}^{L_2} \left(\frac{du_i}{dx} \right)^2 dx$$

Fastener/Contact/Bond Springs

$$U = \frac{1}{2} k (u_i - u_j)^2$$



Mode Decomposition with Single Fastener: Applied Tension Only





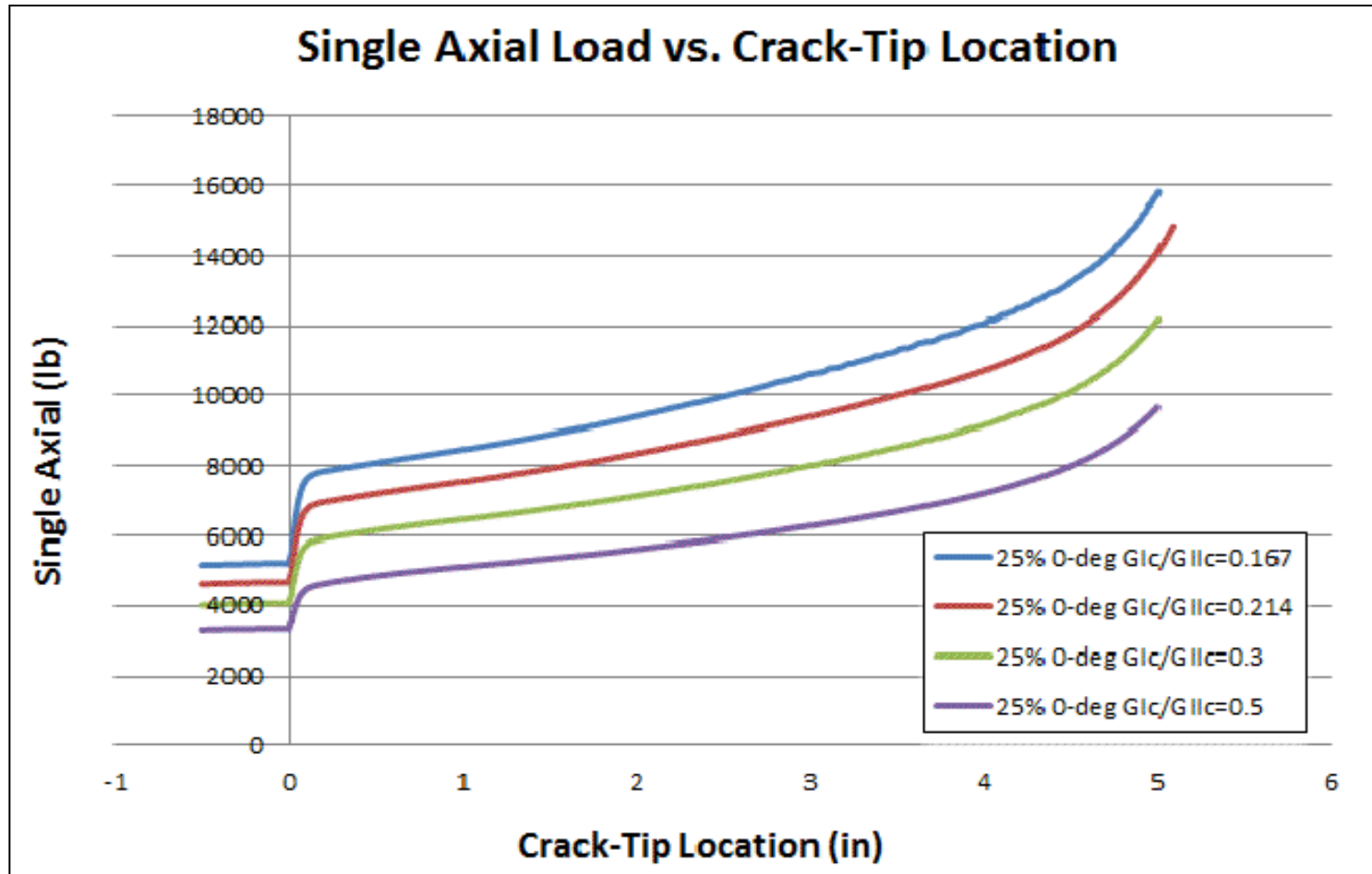
Effects of G_{IC}/G_{IIC} on Crack Propagation

- Assume G_{IC} =constant, but varying G_{IIC}

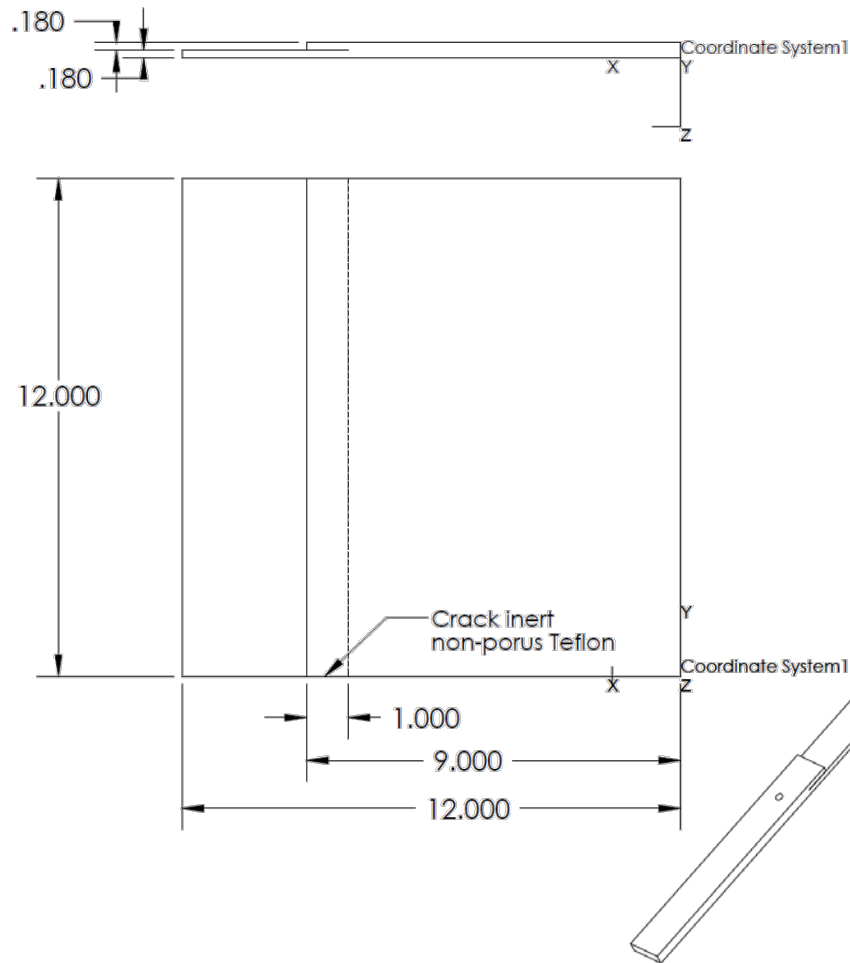
G_{IC} (lb/in)	G_{IIC} (lb/in)	Ratio
1.5	3	0.500
1.5	5	0.300
1.5	7	0.214
1.5	9	0.167
1.5	12	0.125



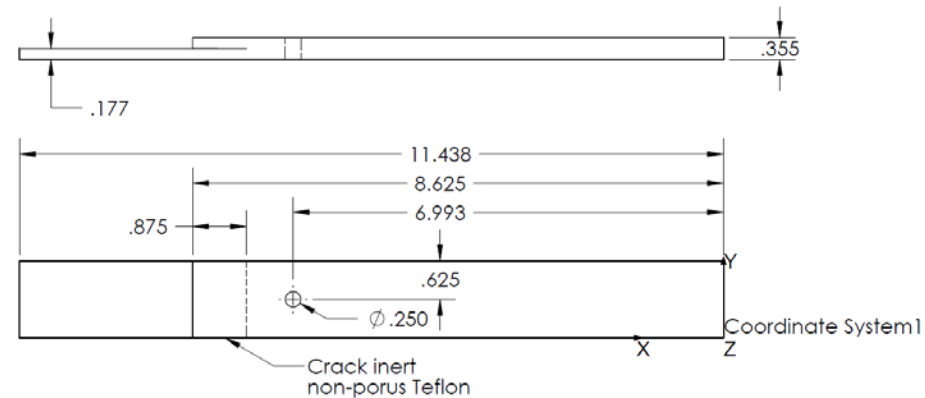
Effects of G_{IC}/G_{IIC} Ratios (Single Fastener)



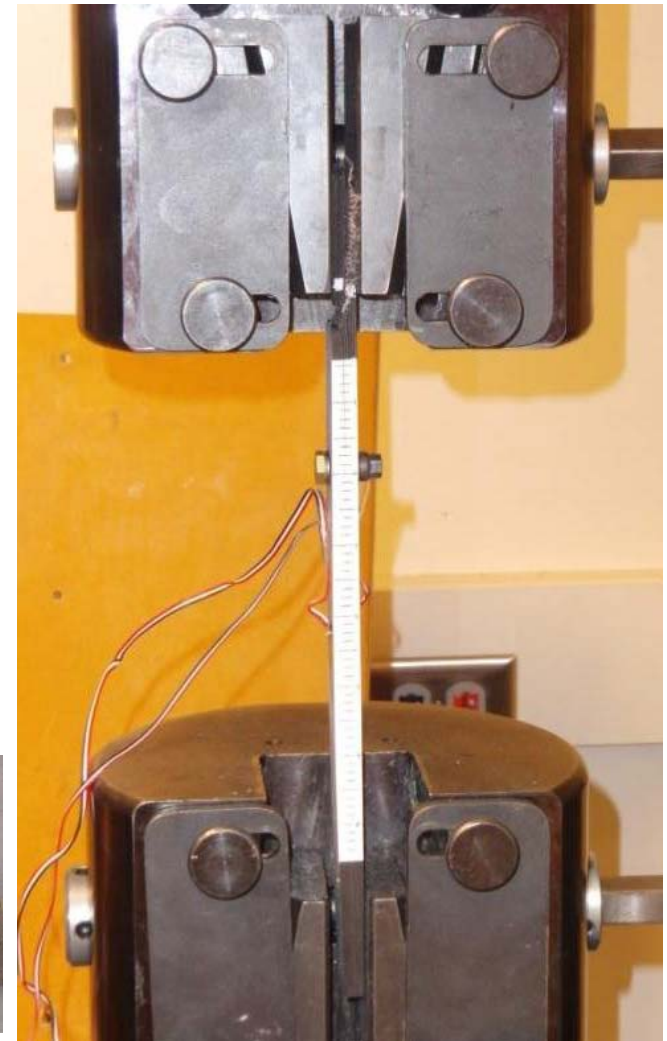
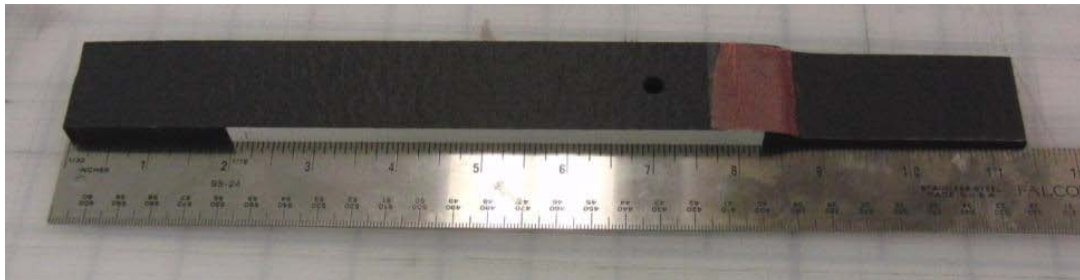
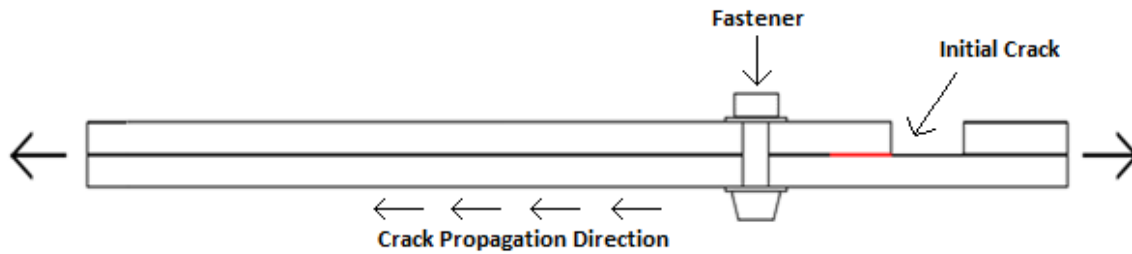
2-Plate Single Fastener Specimen



- BMS 8-276 (T800H/#3900-2) unidirectional pre-preg tape
- BMS 8-308 peel ply
- Titanium fasteners
- $(0/45/90/-45)_3S$
- $(0/-45/02/90/45/02/-45/90/45/0)_S$



2-Plate Single Fastener Specimen



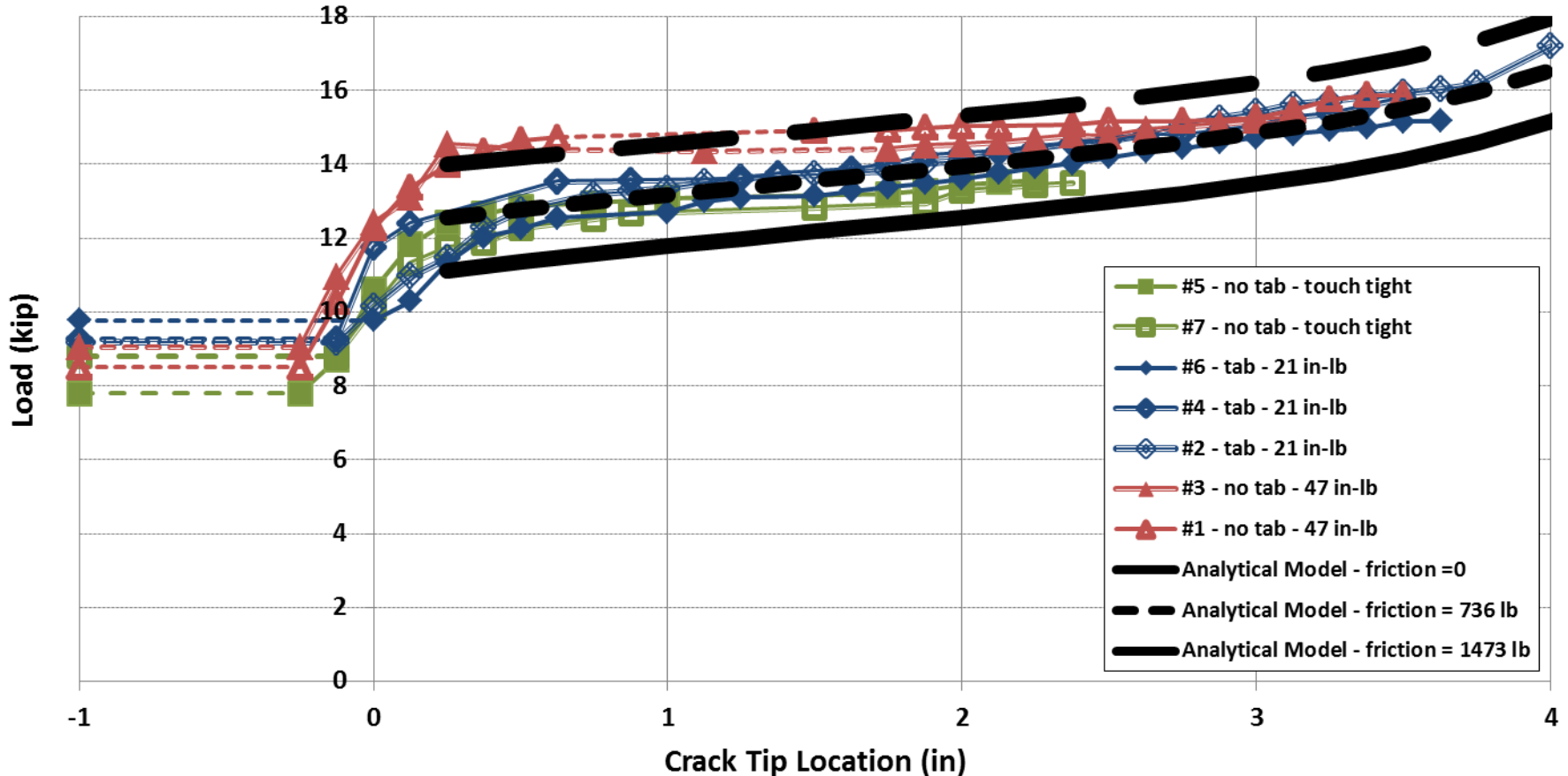


2-Plate Single Fastener Specimen:

$(0/45/90/-45)_{3S}/\text{crack}/(0/45/90/-45)_{3S}$

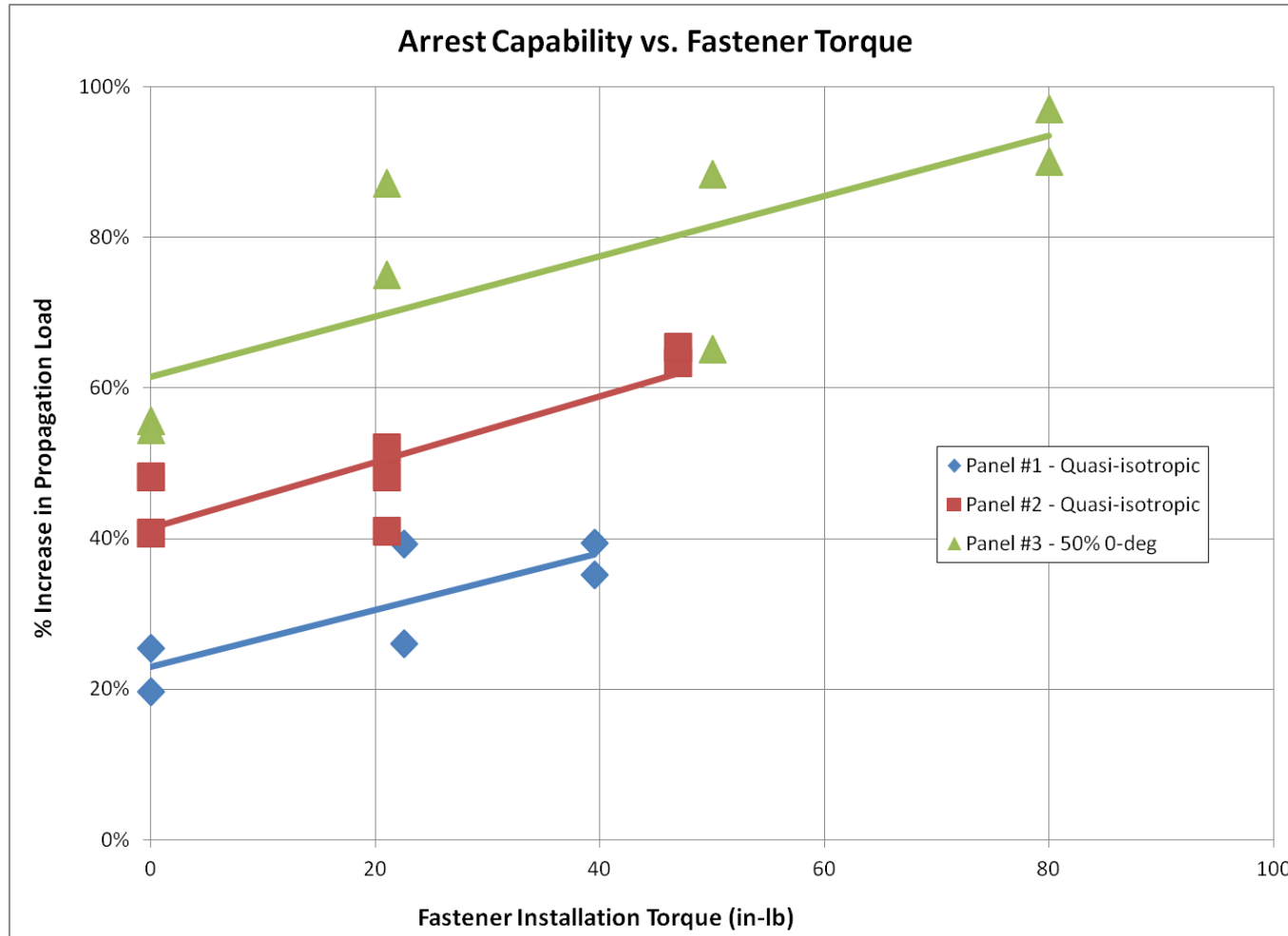
- CLT $E_x = 7.99 \times 10^6$ psi
- Plain Strain $E_x = 8.76 \times 10^6$ psi
- Strain Gauge $E_x = 7.5 \times 10^6$ psi

Load vs. Crack Tip Location
Batch #2 - Quasi-isotropic Lay-up

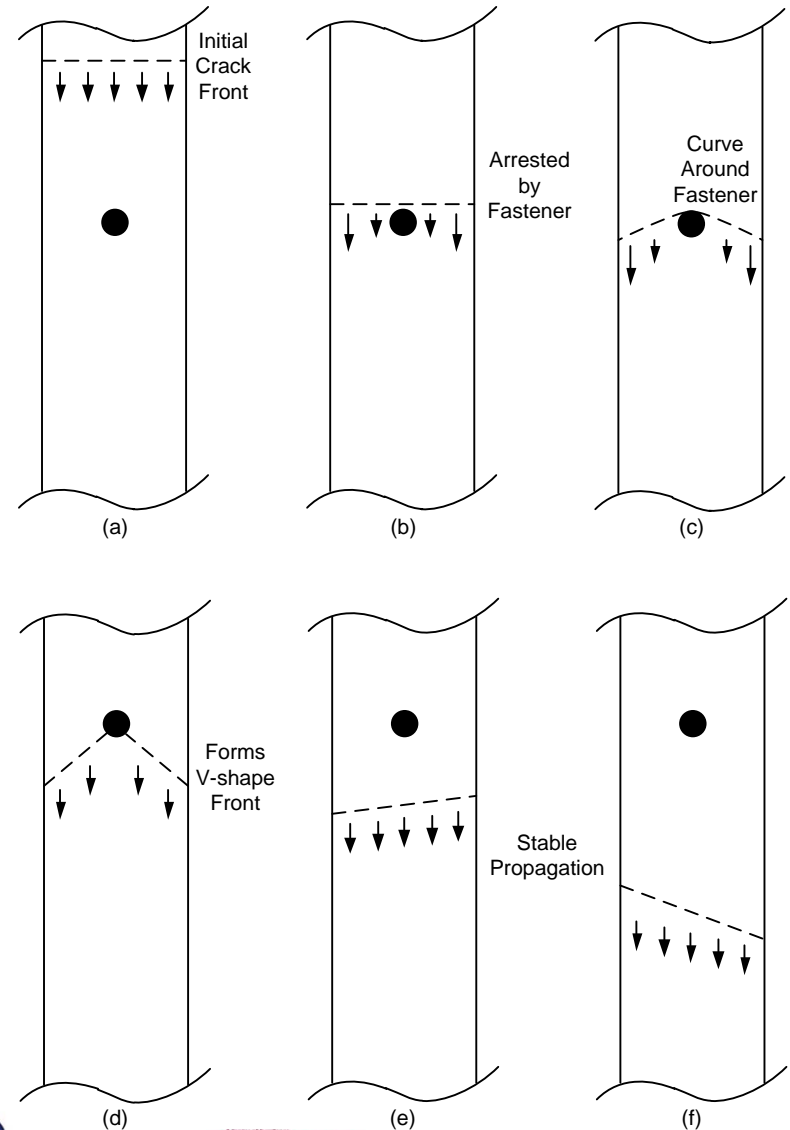
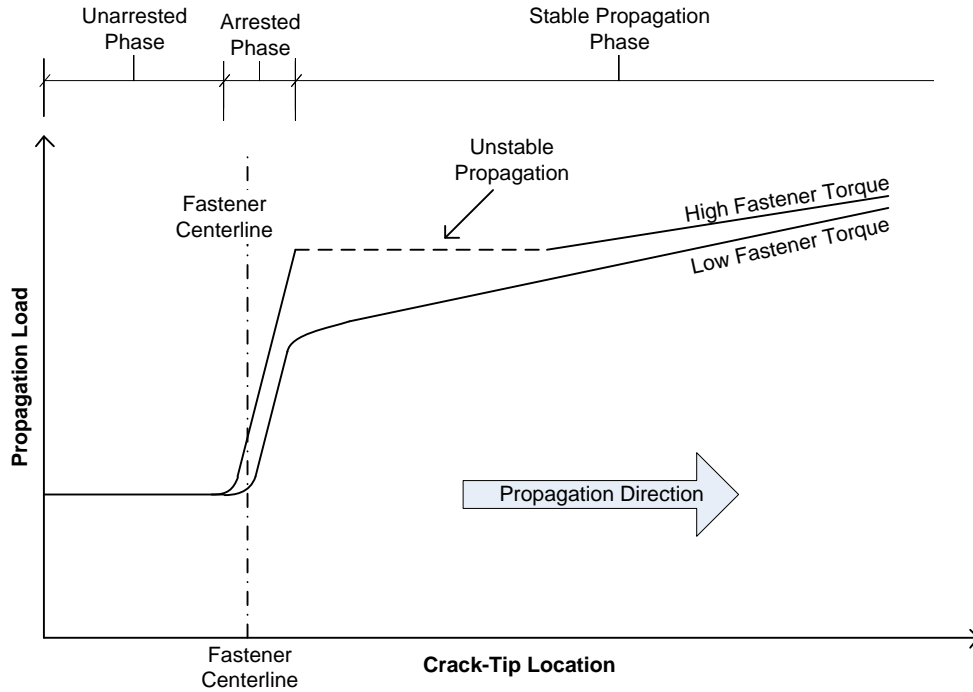




Arrest Capability vs. Fastener Torque



Arrest Mechanisms

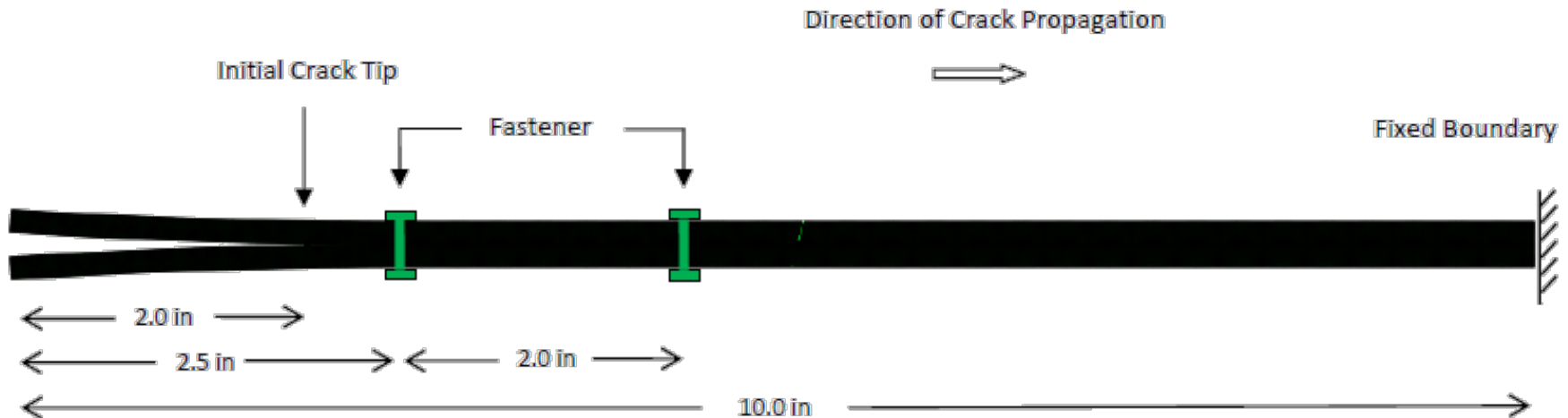


Summary of Single Fastener Results

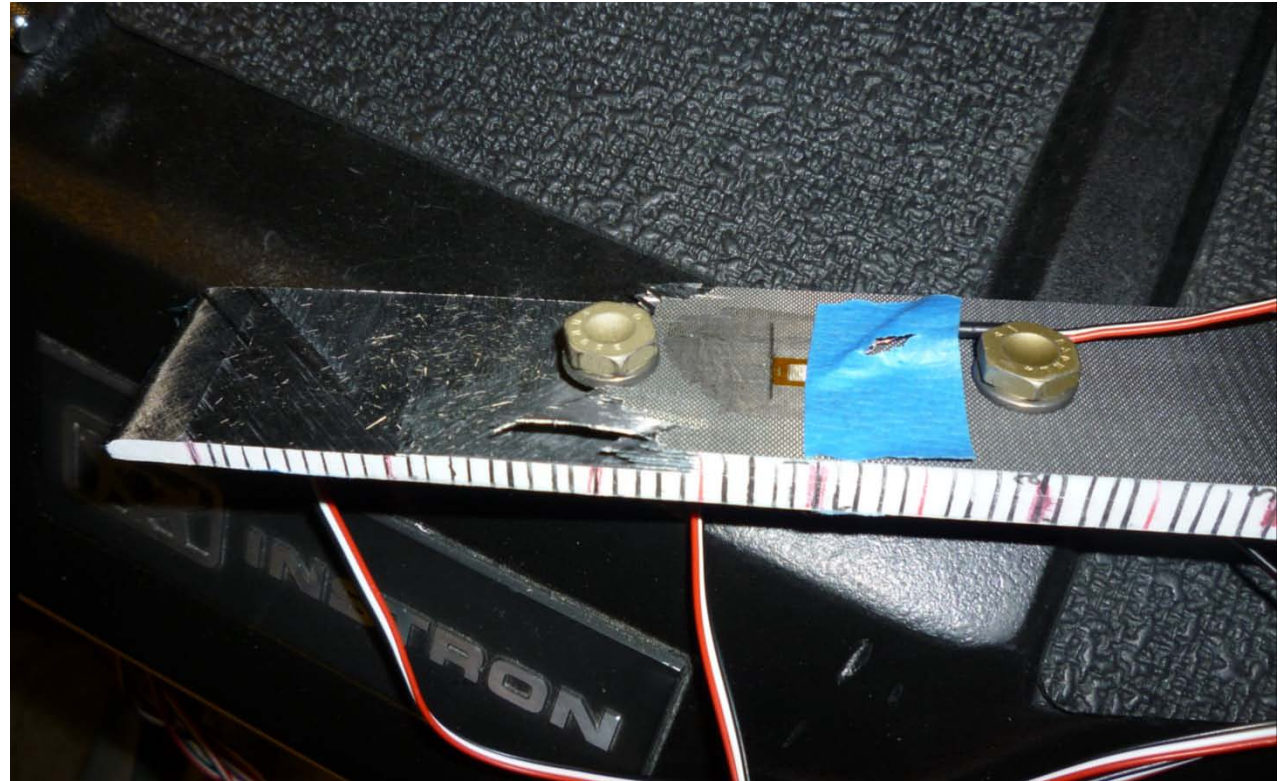
- **Delamination Arrest Mechanism**
 - Mode I suppression
 - Crack-face friction caused by fastener preload
 - Fastener flexibility
- **Limitations**
 - Fasteners do not guarantee arrest in mode II → current research
 - Fastener preload vs. installation torque is hard to control and measure
 - Crack-face friction is poorly understood and rarely studied, even more difficult to model
 - Delamination could steer around the fastener grip

2-Plate Two-Fastener Specimen

- $((0/45/90/-45)_6/\text{Crack})_S$
- Specimen width 1.25"
- T-800S (350°F cure for 2.5hrs)
- 0.25" diameter Ti fasteners, 8D spacing
- Fasteners installed at 40 in-lb (half-torque)
- Holes drilled with aluminum backing
- Load rate 0.1mm/min during crack propagation

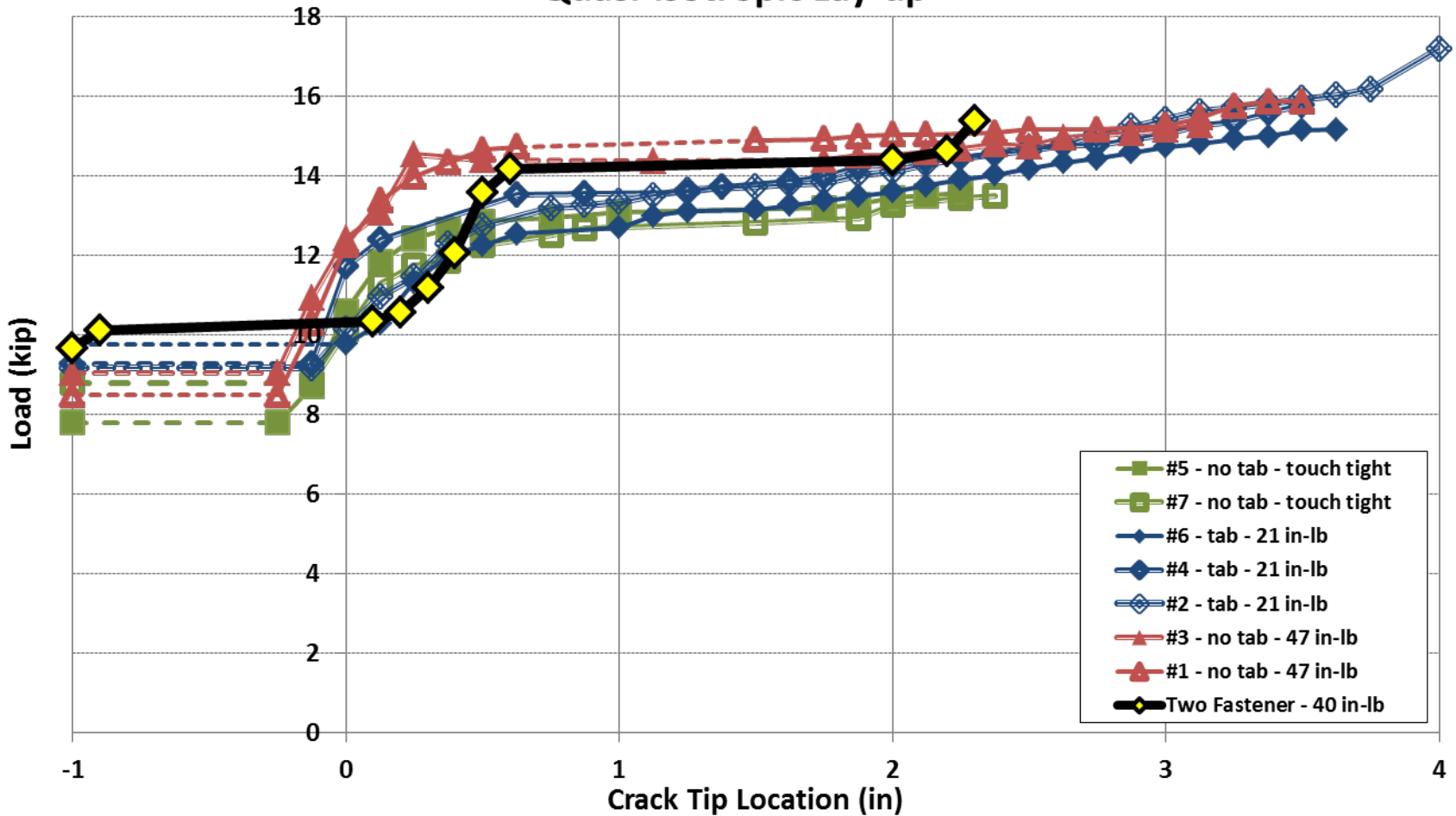


2-Plate Two-Fastener Specimen



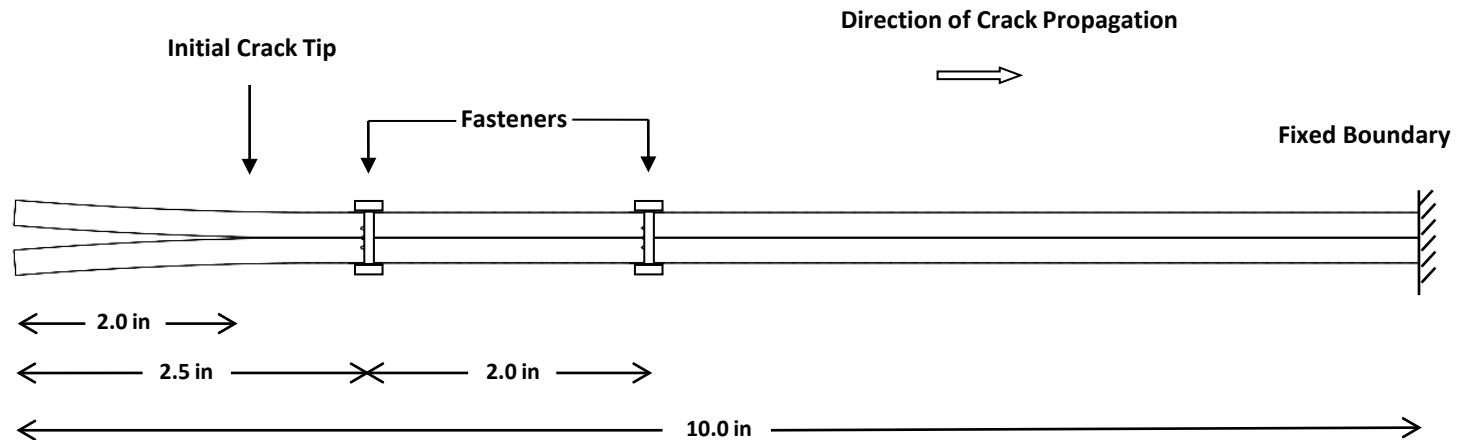
Preliminary Test Results

Load vs. Crack Tip Location Quasi-isotropic Lay-up





2-Plate Two-Fastener FEA Model

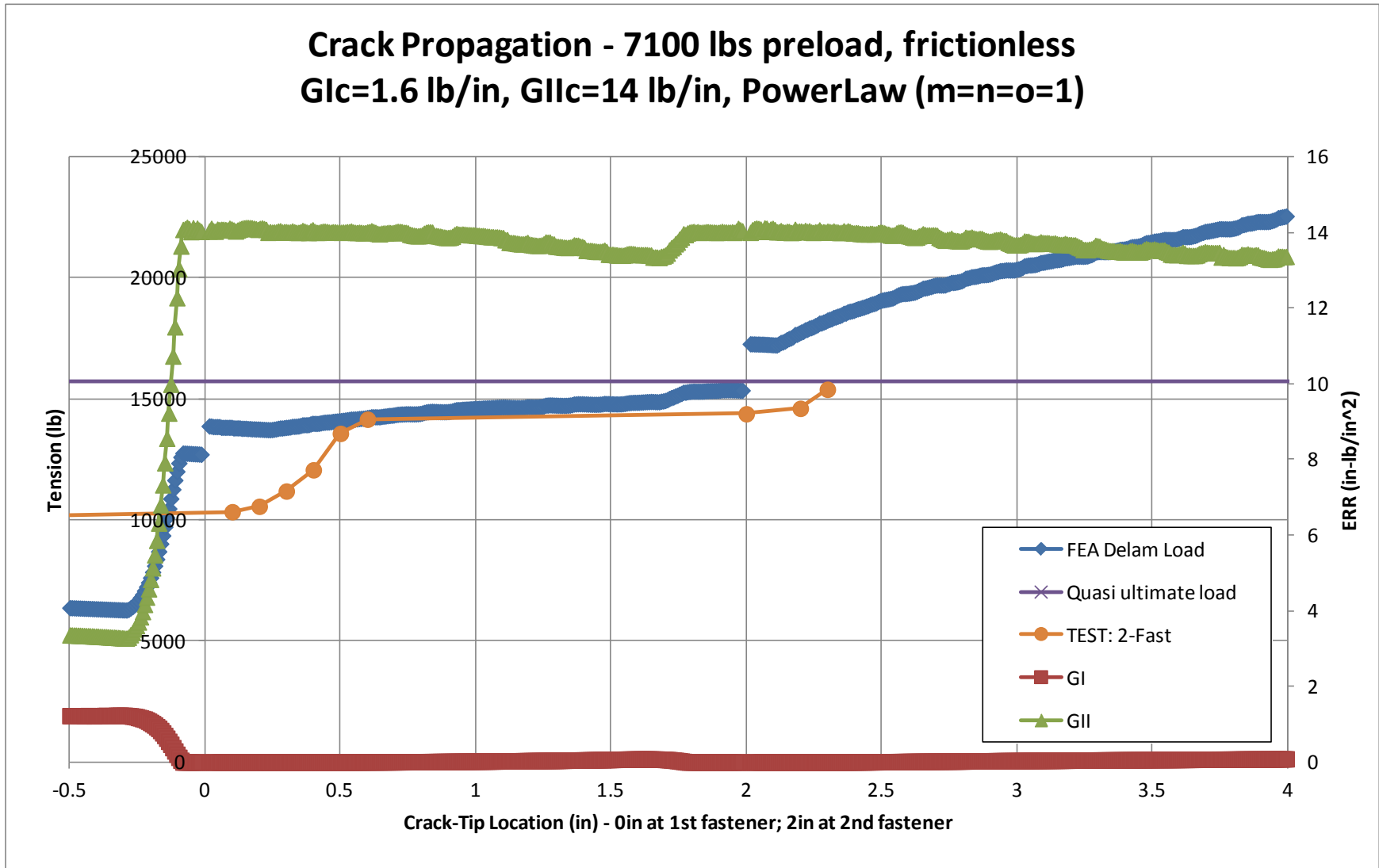


- $((45^\circ/0^\circ/-45^\circ/90^\circ)_3)_s$, ABAQUS CPE4R Element
- Ti-Al6-V4 Fastener, $d = 0.25$ in; T800/3900, $G_{IC}=1.6$ lb/in, $G_{IIC}=14$ lb/in
- Use fastener flexibility (H. Huth, 1986) and fastener tensile stiffness
- B-K law for mixed-mode VCCT criteria



2-Plate Two-Fastener FEA vs. Test Results

Crack Propagation - 7100 lbs preload, frictionless
 $G_{Ic}=1.6 \text{ lb/in}$, $G_{IIc}=14 \text{ lb/in}$, PowerLaw ($m=n=o=1$)



Future Tasks

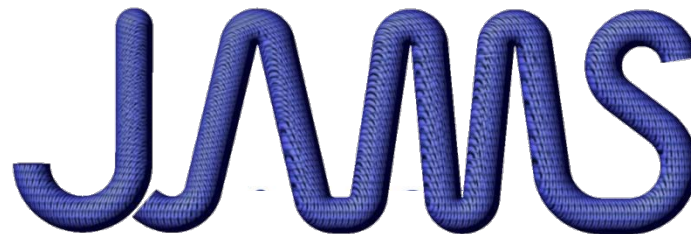
- Finite Element Analysis
 - Design viable 2-fastener specimens
 - Validate model with test results
 - Perform parametric studies on select factors
- Experiment
 - Design viable 2-fastener specimens
 - Manufacture and conduct tests
 - Focus on key factors such as fastener parameters, friction, etc.

Looking Forward

- Benefit to Aviation
 - Tackle one of the main weakness of laminate composite structures
 - Reduce risks (analysis, schedule/cost, re-design, etc.) associated with delamination/disbond mode of failure in large integrated structures
 - Enhance structural safety by building a methodology for designing fail-safe co-cured/bonded structures
- Future needs
 - Initiate research areas core to the interlaminar mode of failure, e.g. friction, fastener clamp-up
 - Industry/regulatory agency inputs related to the application, design, and certification of this type of crack arrest features

End of Presentation

Thank you!



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